

MECH 314

Quiz 1

Fall 2018

16 October 2018

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Problem 1: 30%

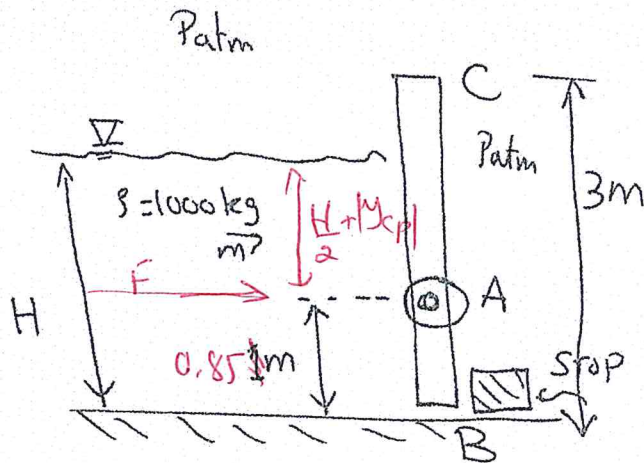
Problem 2: 50%

Problem 3: 20%

$b =$ 1- Vertical gate (BC) is 3 m tall and its mass is 200 kg. It has a width into the page of 1.6 m. The gate is pinned at point (A) and is free to rotate about it. It is kept from opening by the stop at point B. Find the height of water H at which the gate will begin to open. The distance (AB) is 0.85 m. Water has a density of 1000 kg/m³. Assume no friction or vertical reaction force at (B).

Gate starts to open when
 $\Sigma M_A = 0$

Weight of the gate does not contribute to the torque.



$\Rightarrow \Sigma M_A = 0$ when center of pressure $C_p = A$ of water acts on (A)

$$y_{cp} = - \frac{\rho g \sin \theta T_{xx}}{F} \quad ; \quad F = \rho g h_{CG} A$$

$$\Rightarrow F = 9.8 \times 10^3 \times \frac{H}{2} (H \times \overset{1.6m}{b}) = 7.84 \times 10^3 H^2$$

$$y_{cp} = - \frac{9.8 \times 10^3 \sin 90^\circ \times \frac{1}{2} H^3 / 12}{9.8 \times 10^3 \times \frac{1}{2} H^2} = - \frac{H}{6}$$

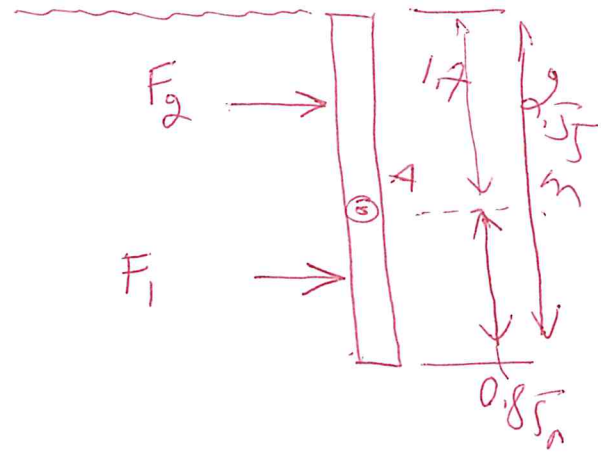
$$\Rightarrow \text{From drawing: } H - \frac{H}{2} - |y_{cp}| = 0.85 \text{ m}$$

$$\Rightarrow H - \frac{H}{2} - \frac{H}{6} = 0.85 \text{ m}$$

$$H = 2.55 \text{ m}$$

Checkery Solution

$$\begin{aligned} F_1 &= \rho g H_{CG} A \\ &= 9.8 \times 10^3 (1.7 + \frac{0.85}{2}) (0.85 \times 1.6) \\ &= 28.322 \text{ kN} \end{aligned}$$



$$y_{cp1} = \frac{\rho g \sin \theta I_{xx}}{F_1}$$

$$y_{cp1} = \frac{\cancel{\rho g} \sin 90 \times \cancel{1.6} \times 0.85^3 / 12}{\cancel{\rho g} (0.85 \times \cancel{1.6}) (1.7 + \frac{0.85}{2})} = 0.0283 \text{ m}$$

$$F_2 = 9.8 \times 10^3 \times \frac{1.7}{2} \times 1.7 \times 1.6 = 22.658 \text{ kN}$$

$$y_{cp2} = \frac{\cancel{\rho g} \sin 90 \times \cancel{1.6} \times (\frac{1.7}{2})^3 / 12}{\cancel{\rho g} (\frac{1.7}{2}) (1.7) (1.6)} = 0.283 \text{ m}$$

$$\begin{aligned} \Sigma M_A &= F_1 \times \perp L_1 + F_2 \times \perp L_2 \\ &= 28.322 \times (\frac{0.85}{2} + 0.0283) - 22.658 \times (1.7 - \frac{1.7}{2} - 0.283) \\ &= 12838 - 12847 \approx \text{Zero} \end{aligned}$$

2- The vertical nozzle shown in the figure delivers 36 m³/hr of water to the atmosphere. It has a mass of 35 kg including the water inside it, with D₁=10 cm and D₂=4 cm. The nozzle is connected at its base at point 1 to a main pipe through flange and bolts. The friction head loss in the nozzle is estimated to be 1.5 m.

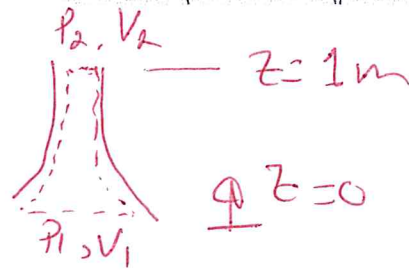
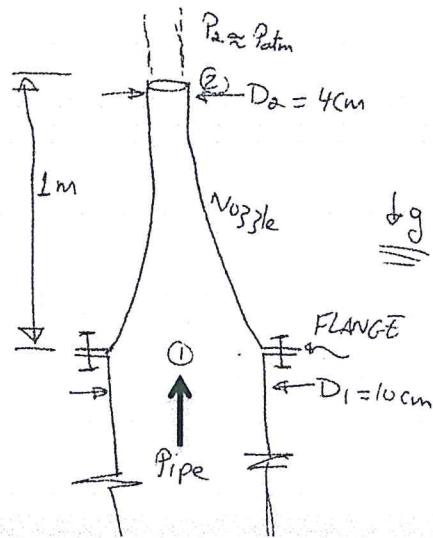
- a- Find the gauge pressure of the water at the base of the nozzle p₁
- b- Find the total force of the flange bolts

$$Q = \frac{36}{3600} = 0.01 \frac{\text{m}^3}{\text{sec}}$$

$$\dot{m} = \rho Q = 10 \text{ kg/sec}$$

$$V_1 = \frac{Q}{\frac{\pi}{4} D_1^2} = 1.27 \frac{\text{m}}{\text{sec}}$$

$$V_2 = \frac{Q}{\frac{\pi}{4} D_2^2} = 7.96 \frac{\text{m}}{\text{sec}}$$



(a) CV as shown

Energy:

$$-h_f = \left(\frac{P_2}{\rho g} + \frac{V_2^2}{2g} + z \right)_2 - \left(\frac{P_1}{\rho g} + \frac{V_1^2}{2g} + z \right)_1$$

gauge

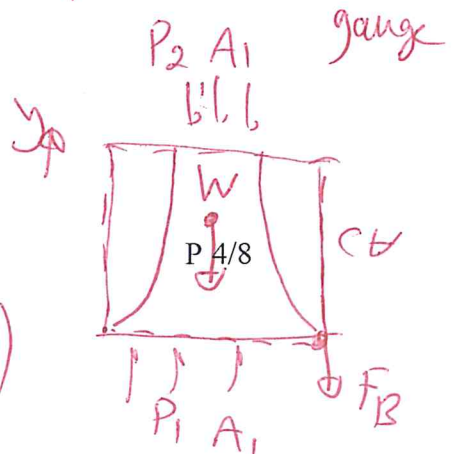
$$-1.5 \text{ m} = \left(\frac{7.96^2}{2 \times 9.8} + 1 \right) - \left(\frac{P_1}{9.8 \times 10^3} + \frac{1.27^2}{2 \times 9.8} + 0 \right)$$

$$-1.5 = 3.23 + 1 - \left(\frac{P_1}{9.8 \times 10^3} + 0.08 \text{ m} \right) \Rightarrow P_1 = 55.37 \text{ kPa}$$

(b) CV as shown $\sum F_y = \dot{m} (V_{y2} - V_{y1})$

$$P_1 A_1 - P_2 A_1 - W - F_B = \dot{m} (V_2 - V_1)$$

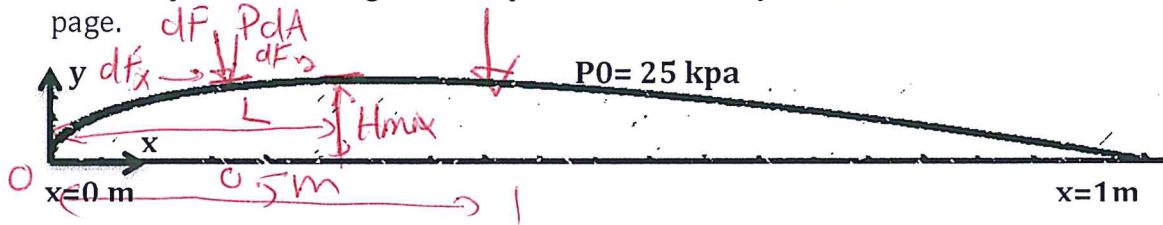
$$55.37 \times 10^3 \times \frac{\pi}{4} (0.1)^2 - 35 \times 9.8 - F_B = 10 (7.96 - 1.27 \frac{\text{m}}{\text{s}})$$



3- The curved body shown in the graph can be described by the equation:

$$y = 5t \left[0.2969\sqrt{x} - 0.1260x - 0.3516x^2 + 0.2843x^3 - 0.1015x^4 \right],$$

with $t=0.15$. Here x extends from 0 to 1 m and y is given in meters. The top side of this curved body is subjected to a uniform gauge pressure p_0 of 25 kPa. You are asked to estimate the **location of the center of pressure** associated with this uniform pressure acting on the top surface. The body has a uniform width into the page.



~~$L_1 = \frac{\sum M_0}{\sqrt{F_x^2 + F_y^2}}$~~

$$L_1 = \frac{\sum M_0}{\sqrt{F_x^2 + F_y^2}}$$

$$F_x = \int_{x=0}^{x=1} dF_x = \int_{y=0}^{y=0} p dy b = p b \int_0^0 dy = 0$$

$$F_y = \int_{x=0}^{x=1} p b dx = p b \Delta x$$

$$\sum M_0 = \int_{x=0}^{x=1} \underbrace{y dF_x}_{\text{ZERO}} + \int x dF_y$$

$$= \int_{x=0}^{x=1} x p b dx = \frac{x^2}{2} p b$$

$$\int y dF_x = \int_{x=0}^{x=1} y p b dy$$

$$= \int_{x=0}^{x=L} y p b dy + \int_{x=L}^{x=0} y p b dy$$

$$= p b \left[\int_0^{H_{max}} y dy + \int_{H_{max}}^0 y dy \right]$$

$$= 0$$

$$L_1 = \frac{\frac{x^2}{2} p b}{p b x} = \frac{x}{2} = 0.5 \text{ m}$$