## Integral Relations for a Control Volume

## 1. (Lecture 1) Control Volume vs System

System: a collection of matter of fixed identity. Control Volume: is a volume fixed in the coordinated frame of reference and is bounded a control surface.

## 2. (Lecture 1) Fundamentals physical laws for a system in primitive form

We are often interested in the rate of change of the amount of a certain physical property contained in a system of volume V. The physical property could be mass, momentum, energy, moment of momentum, entropy etc. The amount of physical property contained in V may be expressed as  $B \equiv \int_V b\rho \, dV$  so that the rate of change is  $\frac{DB}{Dt}$ . The fundamental physical laws governing the rate of change of B is  $\frac{DB}{Dt} = \Phi$ . The table below presents the physical laws of conservation of mass, momentum and energy expressed in the form  $\frac{dB}{dt} = \Phi$ .

Conservation Law	b	В	Φ
mass	1	total mass: $\int_V \rho  dV$	0
linear momentum	u	total momentum: $\int_V \rho \mathbf{u}  dV$	$\sum \mathbf{F}$
angular momentum	$\mathbf{r}  imes \mathbf{u}$	total momentum: $\int_V \rho(\mathbf{r} \times \mathbf{u})  dV$	$\sum \mathbf{M}_O$
energy	$e + \frac{ \mathbf{u} ^2}{2} + U$	total energy: $\int_V \rho\left(e + \frac{ \mathbf{u} ^2}{2} + U\right) dV$	$\sum \delta \dot{q} - \sum \delta \dot{w}$

where  $\rho$  is density, **u** is velocity vector, **r** is position vector, *e* is sepcific internal energy, *U* is potential energy,  $\sum \mathbf{F}$  is the sum of all the forces (surface and body) acting on the control volume,  $\sum \mathbf{M}_O$  is the sum of all the moments about point *O* applied to the control volume,  $\sum \delta \dot{q}$  is the sum of heat gained by the comtrol volume, and  $\sum \delta \dot{w}$  is the sum of work done by the control volume (shaft, pressure, viscous stresses).

## 3. The need for a control volume approach and the Reynolds Transport Theorem

In fluid mechanics, a system undergoes changes in its shape ans volume. It could experience stretching, twisting, expansion etc. It is therefore difficult to predict Vin a Lagrangian sense. It is more convenient to apply the laws of physics for a control volume since it is a fixed volume. In order to apply the above physical laws for a control volume, we need to express the rate of change of the total quantity Bin terms of rate of change inside the control volume and fluxes crossing the control surface. In other words:

$$\frac{D}{Dt} \int_V b\rho \, dV \xrightarrow{?} (?)_{\text{C.V.}}$$

From the figure

$$\frac{DB}{Dt} = \frac{(B_I(t+dt) + B_{II}(t+dt)) - (B_I(t) + B_{III}(t+dt))}{dt} \\
= \frac{B_I(t+dt) - B_I(t)}{dt} + \frac{B_{II}(t+dt)}{dt} - \frac{B_{III}(t+dt)}{dt} \\
= \frac{\partial}{\partial t} B_{\rm CV} + (\text{flux})_{\rm out} - (\text{flux})_{\rm in} \\
= \frac{\partial}{\partial t} \int_{\rm CV} b\rho \, dV + \int_{\rm CS} b \, dw_{\rm out} - \int_{\rm CS} b \, dw_{\rm in}$$

where w is the flow of quantity b through the control surface. Reynolds Transport Theorem

$$\frac{D}{Dt} \int_{\text{system}} b\rho \, dV = \frac{\partial}{\partial t} \int_{\text{CV}} b\rho \, dV + \int_{\text{CS}} b \, (\rho \, \mathbf{u} \cdot d\mathbf{A})$$
System at *t*-*dt*

$$\int_{\text{UV}} \frac{1}{1} \int_{\text{UV}} \frac{1}{1} \int$$

Figure 1: Schematic.

If the control volume (of constant shape) is moving at a constant velocity  $\mathbf{u}_{\rm CV}$ , then Reynolds transport theorem assumes the form

$$\frac{D}{Dt} \int_{\text{system}} b\rho \, dV = \frac{\partial}{\partial t} \int_{\text{CV}} b\rho \, dV + \int_{\text{CS}} b \, \left(\rho \, \mathbf{u}_r \cdot d\mathbf{A}\right)$$

where the relative velocity  $\mathbf{u}_r = \mathbf{u} - \mathbf{u}_{\text{CV}}$ .

- 4. (Lecture 2) Convervation of mass (example)
- 5. (Lecture 2 + Lecture 3) Conservation of linear momentum (example)

- 6. (Lecture 3) Conservation of angular momentum (example)
- 7. (Lecture 4) Conservation of energy (example)

<u>Read</u>: White Sections 3.1-3.6