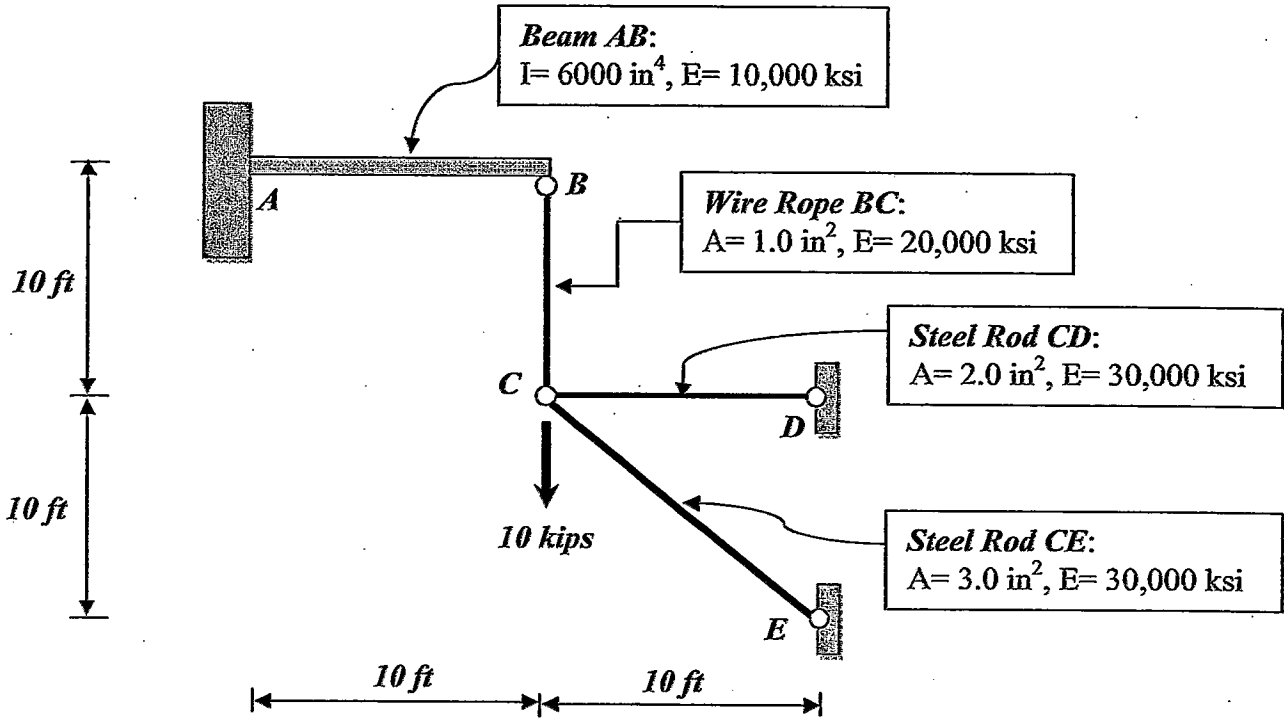


**Problem [1]**

The structure shown in the accompanying figure consists of a beam  $AB$  connected to the axial members  $BC$ ,  $CD$ , and  $CE$ . The composite system is loaded by 10 kips at joint  $C$  as shown. Using the Flexibility Method of Analysis, determine the support reactions at  $A$  and the axial forces in members  $BC$ ,  $CD$ , and  $CE$ .

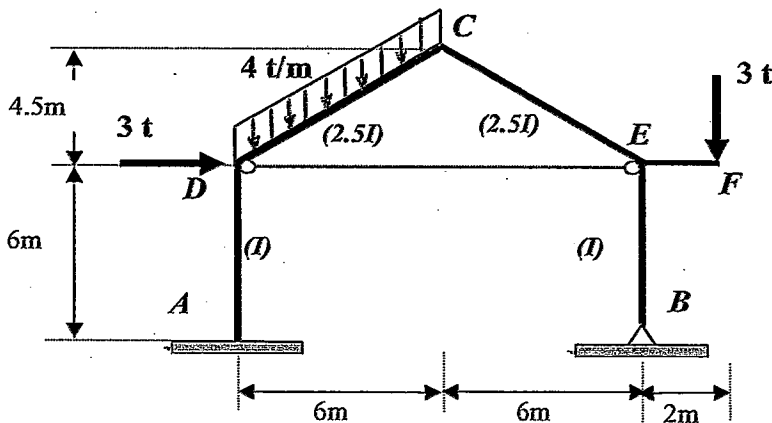
Consider only the effect of bending in  $AB$  (neglect axial and shear deformation), and consider the effect of axial force only in  $BC$ ,  $CD$ , and  $CE$  (neglect flexural deformation).



**Problem [2]**

The frame  $ADCEB$  shown in the accompanying figure is fixed at  $A$  and hinged at  $B$ . The frame is strengthened by a tie  $ED$  as shown. Assemble the flexibility matrix required for analysis of the frame. Neglect axial deformation in all members except member  $DE$ , and Use  $EI = 8000 \text{ t.m}^2$  and  $EA = 20,000 \text{ tons}$  for member  $DE$ .

Use  $X_B$ ,  $Y_B$  and  $T_{DE}$  (the force in the Tie  $DE$ ) as redundants.



### Problem [3]

A plane view of a grid of floor beams, resting on pillars at A, B, C, D, E, and F is shown in the accompanying Figure [a]. Assuming the beam to have identical flexural rigidity,  $EI$ , and zero torsional rigidity:

[1] Using the **Flexibility Method of analysis**, analyze the grid system, given that the segment GH is subjected to a uniformly distributed gravity load of  $20 \text{ kN/m}$  (Figure [b]). Then, draw the shearing Force and bending moment diagrams of beams CD and AB.

[2] Using the **Flexibility Method of analysis**, analyze the grid system, given that joint G is subjected to a concentrated gravity load of  $100 \text{ kN}$  (Figure [c]). Then, draw the shearing Force and bending moment diagrams of beams CD and AB.

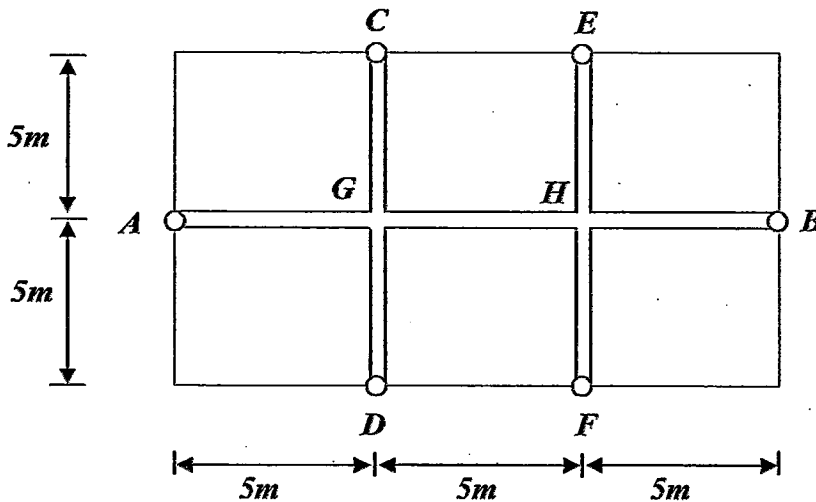


Figure [a]

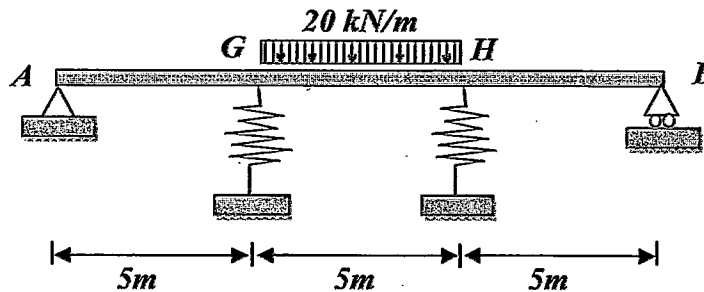


Figure [b]

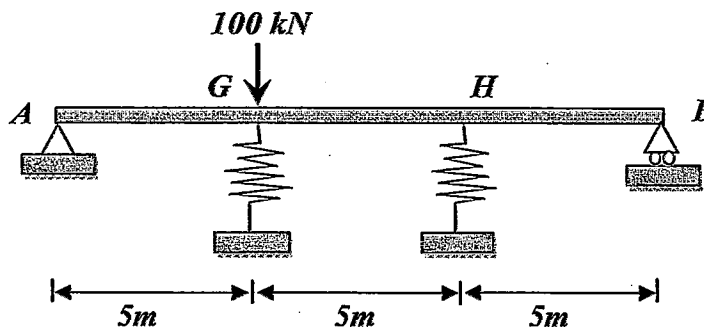
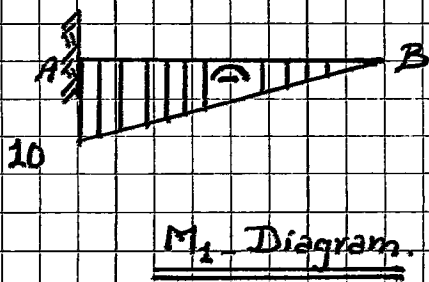
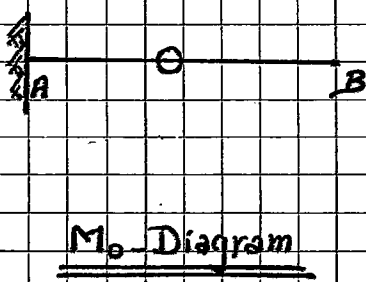
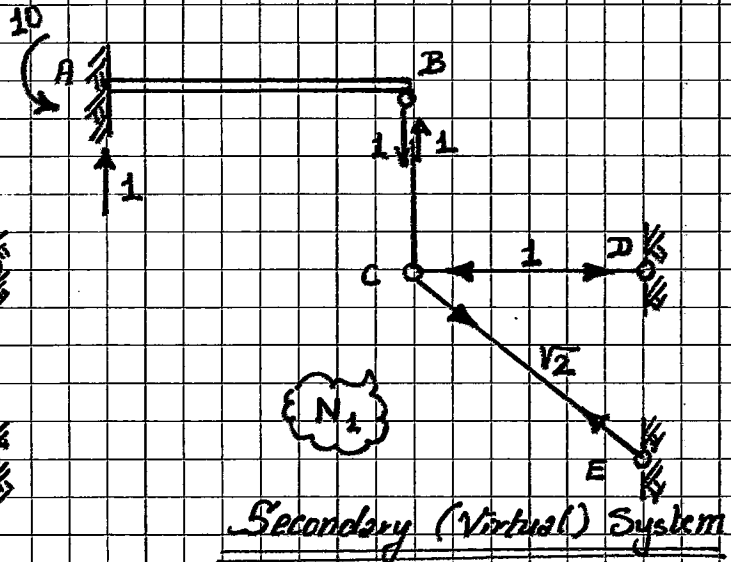
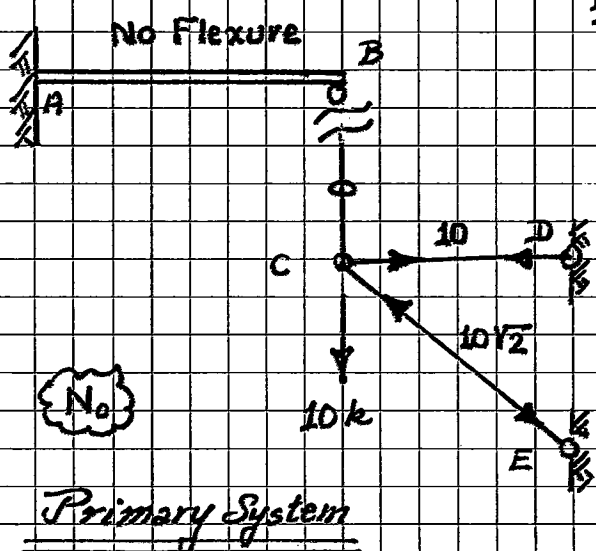
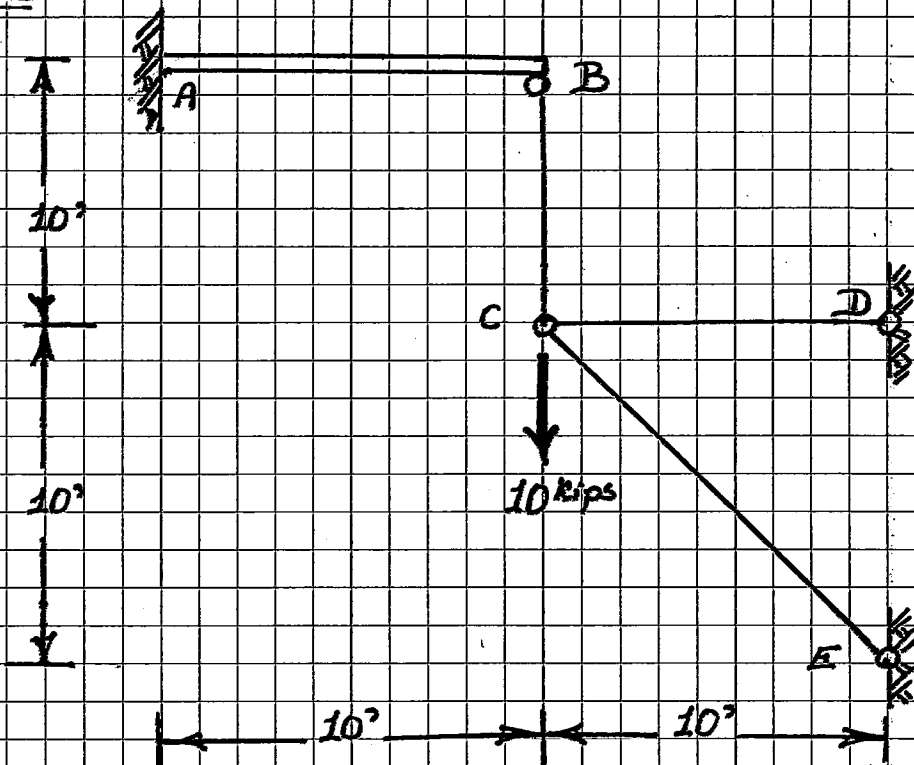


Figure [c]

*Good Luck & Best Wishes  
Dr. Hisham S. Basha, Ph.D.*

PROBLEM 13.



$$f_{10} + x f_{11} = 0$$

$$f_{10} = \int_0^L \frac{M_0 M_1 dl}{EI} + \sum \frac{N_0 N_1 L}{EA} = 0 + \frac{(10)(-1)(10 \times 12)}{30,000 \times 2.0} + \frac{(-10\sqrt{2})(\sqrt{2})(10\sqrt{2})}{30,000 \times 3.0}$$

$$= -0.02 - 0.0377 = -0.0577 \text{ in.}$$

$$f_{11} = \int \frac{M_1 M_1 dl}{EI} + \sum \frac{N_1 N_1 L}{EA} = \frac{10 \times 12^3 (10 \times 10)}{3EI} + \frac{(1)(1)(10 \times 12)}{20,000 \times 1.0}$$

$$+ \frac{(1)(1)(10 \times 12)}{30,000 \times 2.0} + \frac{(\sqrt{2})(\sqrt{2})(10\sqrt{2} \times 12)}{30,000 \times 3.0}$$

$$\Rightarrow f_{11} = \frac{10 \times (10 \times 10) \times 12^3}{3 \times 10,000 \times 6000} + 0.006 + 0.002 + 0.00377$$

$$= 0.0096 + 0.006 + 0.002 + 0.00377 = 0.02137 \text{ in.}$$

$$\text{Now, } x = \frac{f_{10}}{f_{11}} = \frac{-0.0577}{0.02137} = \underline{\underline{+2.70 \text{ k.}}}$$

$x$  = Force in Wire Rope BC, the +ve sense indicates force is in the assumed sense of the dummy load (Tension).

$$\text{Now, } R_{\text{Final}} = R_0 + x R_1$$

$$\Rightarrow F_{CD} = 10 + (2.70)(-1) = \underline{\underline{+7.30 \text{ k}}} \text{ (Tension)}$$

$$F_{CE} = -10\sqrt{2} + (2.70)(\sqrt{2}) = \underline{\underline{-10.32 \text{ k}}} \text{ (Compression)}$$

$$Y_A = 0 + (2.70)(1) = \underline{\underline{2.70 \text{ k}}} \text{ (}\uparrow\text{)}$$

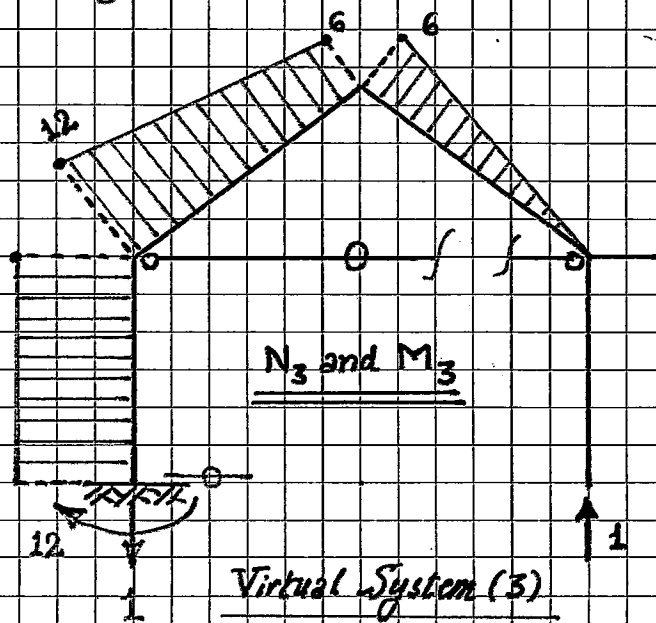
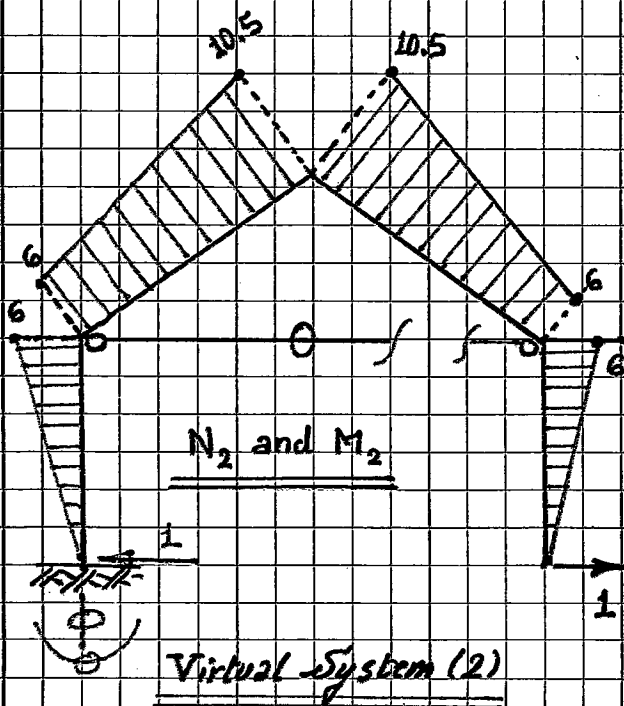
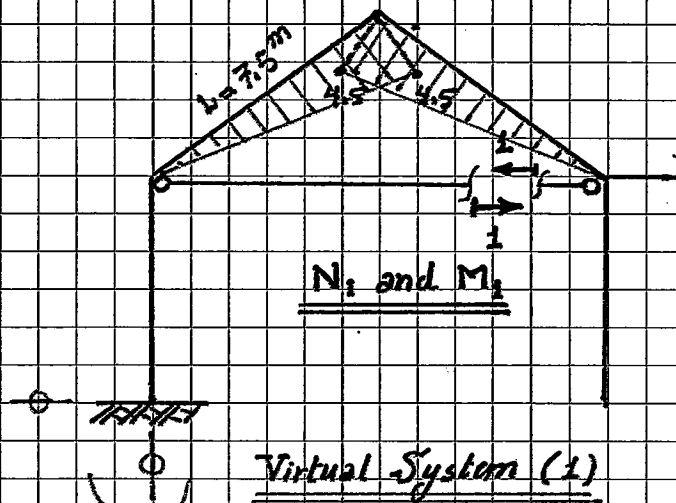
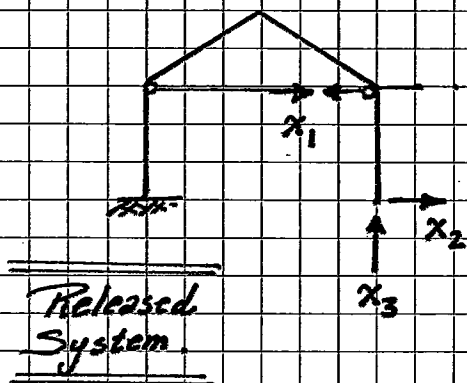
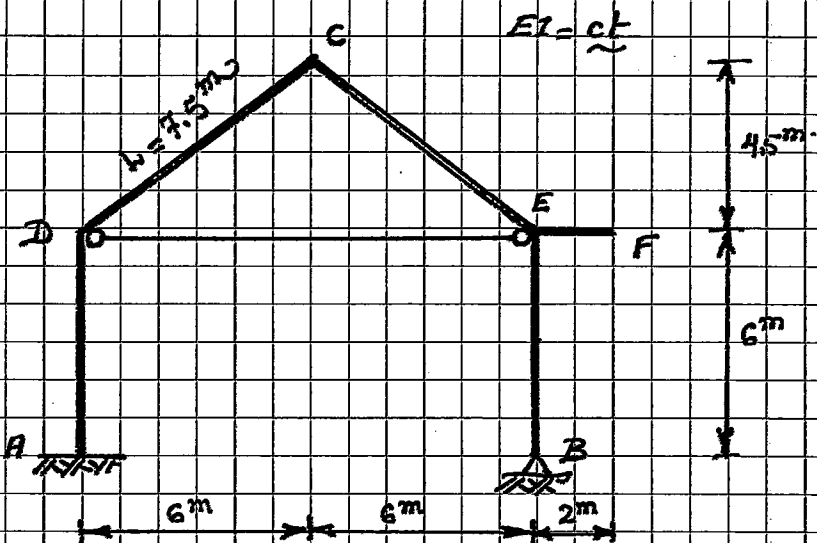
$$M_A = 0 + (2.70)(-10) = \underline{\underline{-27 \text{ k.ft}}} \text{ (}\curvearrowright\text{)}$$

PROBLEM {23}

System is indet. to the third degree.

$\rightarrow [F]_{3 \times 3}$

$$[F] = \begin{bmatrix} f_{11} & f_{12} & f_{13} \\ f_{21} & f_{22} & f_{23} \\ f_{31} & f_{32} & f_{33} \end{bmatrix}_{3 \times 3}$$



## Coefficients of the Flexibility Matrix.

$$f_{11} = \int \frac{M_1 M_1 dl}{EI} + \sum \frac{N_1 N_1 L}{EA}$$

$$f_{21} = f_{12} = \checkmark$$

$$f_{12} = \int \frac{M_1 M_2 dl}{EI} + \sum \frac{N_1 N_2 L}{EA} \rightarrow 0$$

$$f_{31} = f_{13} = \checkmark$$

$$f_{13} = \int \frac{M_1 M_3 dl}{EI} + \sum \frac{N_1 N_3 L}{EA} \rightarrow 0$$

$$f_{23} = f_{32} = \checkmark$$

$$f_{22} = \int \frac{M_2 M_2 dl}{EI} + \sum \frac{N_2 N_2 L}{EA} \rightarrow 0$$

$$f_{23} = \int \frac{M_2 M_3 dl}{EI} + \sum \frac{N_2 N_3 L}{EA} \rightarrow 0$$

Note:  $EI = 8000 \text{ t.m}^2$   
 $EA = 20,000 \text{ t}$   
 $\Rightarrow EA \approx 2.5 EI$

$$f_{33} = \int \frac{M_3 M_3 dl}{EI} + \sum \frac{N_3 N_3 L}{EA} \rightarrow 0$$

$$\bullet f_{11} = \frac{7.5}{3EI} [4.5 \times 4.5] \times 2 + \frac{(1)(1)(12)}{EA} = \frac{106.05}{EI}$$

$$\bullet f_{12} = \frac{7.5}{6EI} \{ 0 + 2(10.5)(-4.5) + (6)(-4.5) + 0 \} \times 2 = -\frac{303.75}{EI}$$

$$\bullet f_{13} = \frac{7.5}{6EI} \{ 0 + 2(6)(-4.5) + (12)(-4.5) + 0 \} + \frac{7.5}{3EI} (-6)(4.5) = -\frac{202.5}{EI}$$

$$\bullet f_{22} = 2 \times \left\{ \frac{7.5}{6EI} [2(6 \times 6) + 2(10.5)(10.5) + (6)(10.5) + (10.5)(6)] + \frac{6}{3EI} (6 \times 6) \right\} = \frac{1190.25}{EI}$$

$$\bullet f_{23} = \frac{1}{EI} (12 \times 6 \times 3) + \frac{7.5}{6EI} [2(6 \times 12) + 2(10.5 \times 6) + (6 \times 6) + (12 \times 10.5)] +$$

$$\frac{7.5}{6EI} [0 + 2(6 \times 10.5) + 0 + (6 \times 6)] = \frac{958.5}{EI}$$

$$\bullet f_{33} = \frac{1}{EI} (12 \times 6 \times 12) + \frac{7.5}{6EI} [2(12)(12) + 2(6 \times 6) + 12 \times 6 + 6 \times 12] + \frac{7.5}{3EI} (6 \times 6)$$

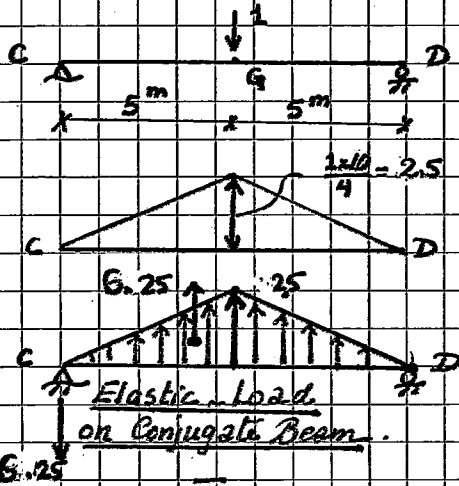
$$= \frac{1584}{EI}$$

$$\Rightarrow [F]_{3 \times 3} = \begin{bmatrix} 106.05 & -303.75 & -202.5 \\ -303.75 & 1190.25 & 958.5 \\ -202.5 & 958.5 & 1584 \end{bmatrix} \times \frac{1}{EI} = \begin{bmatrix} 0.01325 & -0.038 & -0.025 \\ -0.038 & 0.1488 & 0.1198 \\ -0.025 & 0.1198 & 0.198 \end{bmatrix}$$

**PROBLEM {3}**

[17].

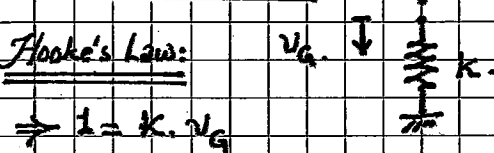
Beams CD and EF have identical stiffnesses. (K).



$$v_G = \frac{M}{EI} = \left[ 6.25(5) + 6.25\left(\frac{5}{3}\right) \right] \frac{1}{EI}$$

$$v_G = -\frac{20.833}{EI} \quad (\downarrow)$$

⇒ Equiv. linear Spring of Axial Stiffness (K).



Hook's Law:

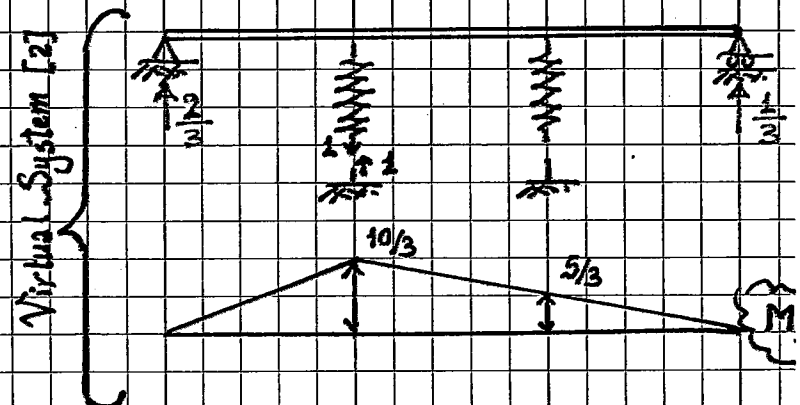
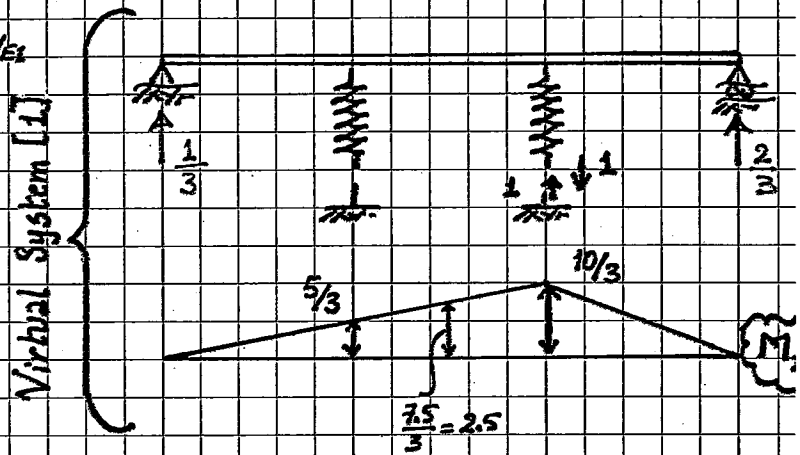
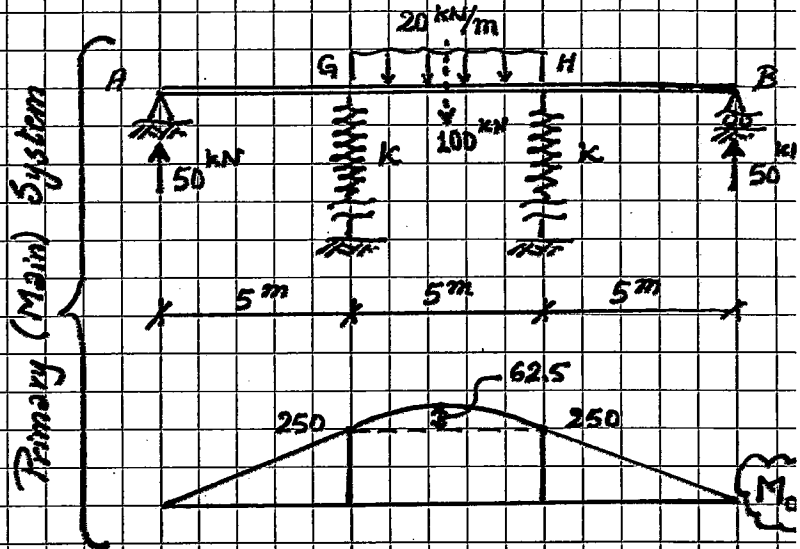
$$\Rightarrow l = K \cdot v_G$$

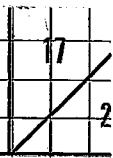
$$\Rightarrow K = \frac{l}{v_G} = \frac{EI}{20.833}$$

OR  $\Delta_c = \frac{PL^3}{48EI} \Rightarrow K = \frac{P}{\Delta} = \frac{48EI}{L^3}$   
 $\Rightarrow K = \frac{EI}{20.833}$  (same)

$$\begin{Bmatrix} f_{10} \\ f_{20} \end{Bmatrix} + \begin{bmatrix} f_{11} & f_{12} \\ f_{21} & f_{22} \end{bmatrix} \begin{Bmatrix} x_1 \\ x_2 \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \end{Bmatrix}$$

Flex. Matrix





$$\begin{aligned}
 f_{10} &= \int \frac{M_0 M_1 dl}{EI} + \sum \frac{N_0 N_1 L}{EA} = \int \frac{M_0 M_1 dl}{EI} + \sum \left( N_0 N_1 \times \frac{1}{K} \right) \\
 &= \int_A^G \frac{M_0 M_1 dl}{EI} + \int_G^H \frac{M_0 M_1 dl}{EI} + \int_H^B \frac{M_0 M_1 dl}{EI} + \sum \left( N_0 N_1 \times \frac{1}{K} \right) \\
 &= \frac{5}{3EI} \left( \frac{5}{3} \times 250 \right) + \frac{1}{EI} \left[ (250 \times 5)(2.5) + \frac{2}{3} (62.5)(5)(2.5) \right] + \frac{5}{3EI} \left( \frac{10}{3} \times 250 \right) + \text{Zero} \\
 &= \frac{(694.44 + 3,645.83 + 1,388.89)}{EI} = \frac{5,729.16}{EI}
 \end{aligned}$$

$f_{20} = f_{10}$  --- Due to symmetry.

$$f_{11} = \frac{10}{3EI} \left( \frac{10}{3} \times \frac{10}{3} \right) + \frac{5}{3EI} \left( \frac{10}{3} \times \frac{10}{3} \right) + \frac{(1)(1)}{EI} \left( \frac{20.833}{EI} \right) = \frac{76.388}{EI}$$

$f_{22} = f_{11}$  --- Due to symmetry.

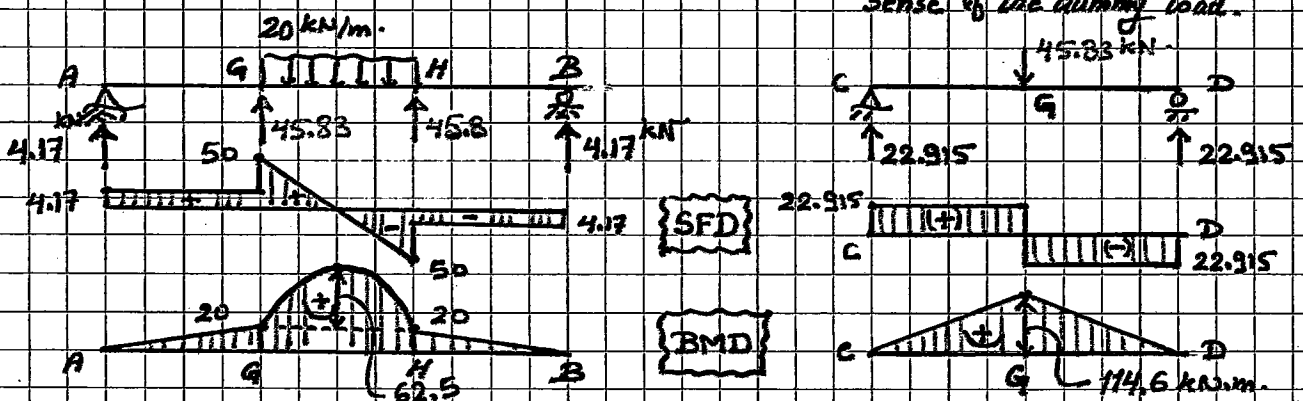
$$\begin{aligned}
 f_{12} = f_{21} &= \int \frac{M_1 M_2 dl}{EI} + \sum \left( N_1 N_2 \times \frac{1}{K} \right) \\
 &= \frac{5}{3EI} \left( \frac{10}{3} \times \frac{5}{3} \right) \times 2 + \frac{5}{6EI} \left[ 2 \left( \frac{5}{3} \right) \left( \frac{10}{3} \right) + 2 \left( \frac{10}{3} \right) \left( \frac{5}{3} \right) + \left( \frac{5}{3} \right) \left( \frac{5}{3} \right) + \left( \frac{10}{3} \right) \right] \\
 &\quad + \text{Zero} + \text{Zero} \\
 &= \frac{18.5185}{EI} + \frac{30.093}{EI} = \frac{48.611}{EI}
 \end{aligned}$$

$$\Rightarrow \frac{1}{EI} \begin{Bmatrix} 5,729.16 \\ 5,729.16 \end{Bmatrix} + \frac{1}{EI} \begin{bmatrix} 76.388 & 48.611 \\ 48.611 & 76.388 \end{bmatrix} \begin{Bmatrix} X_1 \\ X_2 \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \end{Bmatrix}$$

Due to symmetry  $X_1 = X_2 = X$  (Force in the spring).

$$\Rightarrow X = - \frac{5729.16}{125} = \underline{\underline{-45.83 \text{ kN}}} \quad (\text{-ve} \Rightarrow \text{Comp. in the spring})$$

opp. to assumed sense of the dummy load.





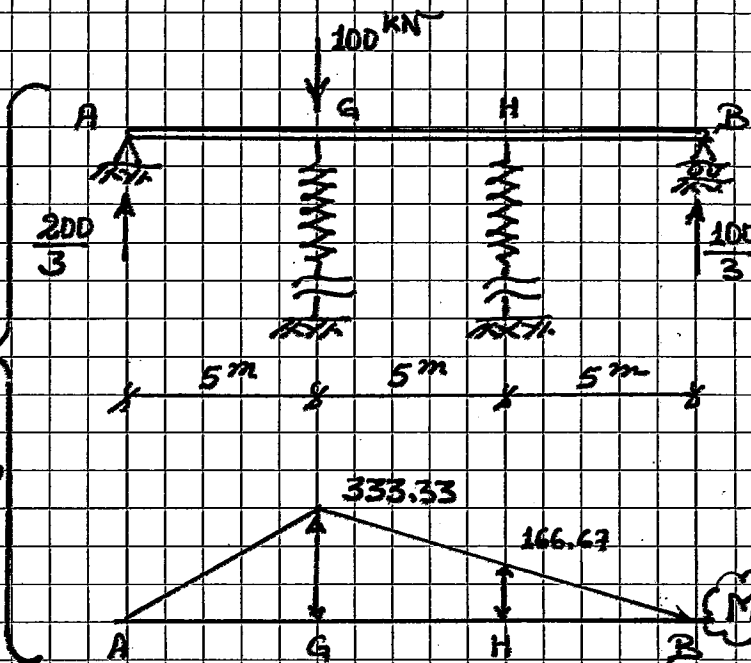
[2].

$$\begin{Bmatrix} f_{10} \\ f_{20} \end{Bmatrix} = ?$$

$$f_{10} = \int \frac{M_0 M_1 dl}{EI} + \sum (N_0 N_1 \frac{1}{K})$$

$$f_{20} = \int \frac{M_0 M_2 dl}{EI} + \sum (N_0 N_2 \frac{1}{K})$$

Primary (Main) system



$$\begin{aligned} \Rightarrow f_{10} &= \frac{5}{3EI} (333.33 \times 5) + \frac{5}{6EI} [2(333.33)(\frac{5}{2}) \\ &\quad + 2(166.67)(\frac{10}{3}) + (333.33)(\frac{10}{3}) + (166.67)(\frac{5}{2})] \\ &\quad + \frac{5}{3EI} (\frac{10}{3})(166.67) = \frac{1}{EI} (925.92 + 3009.26 + 925.94) \\ &= \frac{4861.12}{EI} \end{aligned}$$

Note:

Virtual system [1] → As before

Virtual system [2] → As before

⇒  $\begin{Bmatrix} f \end{Bmatrix}_{2 \times 2}$  as before.

$$\begin{aligned} f_{20} &= \frac{5}{3EI} (10 \times 333.33) + \frac{5}{6EI} [2(\frac{10}{3})(333.33) + \\ &\quad 2(166.67)(\frac{5}{2}) + (10)(166.67) + (\frac{5}{3})(333.33)] \\ &\quad + \frac{5}{3EI} (\frac{5}{2})(166.67) \\ &= (1851.83 + 3240.74 + 462.97) \frac{1}{EI} \\ &= \frac{5555.6}{EI} \end{aligned}$$

$$\Rightarrow \frac{1}{EI} \begin{Bmatrix} 4861.12 \\ 5555.6 \end{Bmatrix} + \begin{bmatrix} \frac{76.388}{EI} & \frac{48.611}{EI} \\ \frac{48.611}{EI} & \frac{76.388}{EI} \end{bmatrix} \begin{Bmatrix} x_1 \\ x_2 \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \end{Bmatrix}$$

Solving ⇒  $x_1 = \underline{\underline{-29.17 \text{ kN}}}$  and  $x_2 = \underline{\underline{-54.17 \text{ kN}}}$   
 comp. in the spring

Plot S.F.D & BMD as usual.