

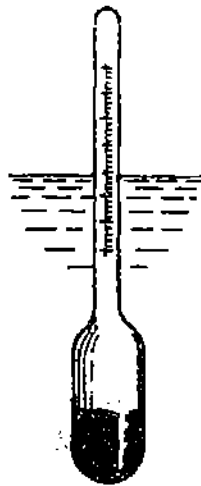
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MEN 330  
EXAM 1  
FALL 2006

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- 1) (15 pts) The hydrometer shown is used to measure the specific gravity of liquids. The mass of the float is  $0.0372 \text{ kg}$  and the diameter of the cylindrical section protruding above the surface is  $0.0064 \text{ m}$ . Determine the period of oscillation of the float in a liquid of specific gravity 1.20. Note that the specific weight of a liquid (in units of  $\frac{\text{force}}{\text{volume}}$ ) is equal to the specific gravity of the liquid multiplied by  $9806 \frac{\text{N}}{\text{m}^3}$ .

$m_f = 0.0372 \text{ kg}$   
 $d = 0.0064 \text{ m}$   
 $sg = 1.2$   
specific weight  $\gamma$   
 ~~$\gamma = sg \times 9806$~~   $\gamma = sg \times 9806$



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mass of the hydrometer is  $m_f = 0.0372 \text{ kg}$ .

Restoring force in the liquid:  $F = \rho g \times V_{\text{displaced}}$

$$\text{units of } \rho g = \frac{\text{kg}}{\text{m}^3} \times \frac{\text{m}}{\text{s}^2} = \frac{\text{kg}}{\text{m}^2 \text{s}^2} = \frac{\text{kg} \cdot \text{m}}{\text{s}^2 \cdot \text{m}^3} = \frac{\text{N}}{\text{m}^3} \Rightarrow \rho g = \gamma$$

$$F = \gamma \times V_{\text{displaced}} = 1.2 \times 9806 \times A \times x \quad \checkmark$$

$$= 1.2 \times 9806 \times \pi r^2 \times x$$

$$= 1.2 \times 9806 \times \pi (0.0032)^2 \times x$$

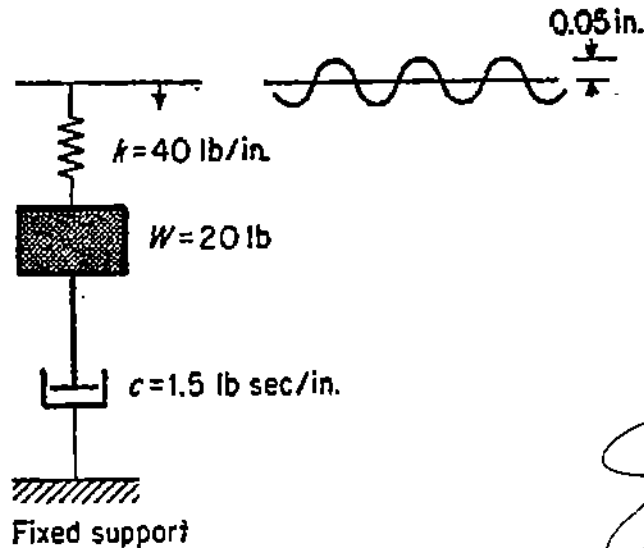
$$\Rightarrow 0.37855 x \quad \checkmark$$

$$\Rightarrow 0.0372 \ddot{x} + 0.37855 x = 0 \quad \checkmark$$

$$\omega = \sqrt{\frac{0.37855}{0.0372}} = 3.19 \text{ rad/s} \quad \checkmark$$

$$T = \frac{2\pi}{\omega} = 1.97 \text{ s}$$

- 2) (20 pts) The upper support of the system shown moves harmonically with an amplitude of 0.05 in and with a frequency equal to the undamped natural frequency of the system. Calculate the amplitudes of the spring and damping forces.



*Handwritten scribble*

$k = 40 \text{ lb/in}$      $W = 20 \text{ lb}$      $c = 1.5 \text{ lb sec/in}$

$y(t) = \frac{1}{2} \sin \omega t$

$\omega = \omega_n$   
 $\Rightarrow r = 1$

$y(t) = 0.05 \sin \omega t$

equation of motion:  
 $k(x-y) + c(\dot{x}-\dot{y}) - m\ddot{x} = \frac{1}{2} \sin \omega t$

*Handwritten notes:*  
 $c < 2m\omega_n$   
 $2 \times 20 \times 27.8$   
 $1112$

$\omega = \omega_n = \sqrt{\frac{k}{m}} = \sqrt{\frac{40}{20/386.4}} = 27.8 \text{ rad/s}$

$r = \frac{c}{2m\omega_n} = \frac{1.5}{2 \times 20 \times 27.8} = 0.52$

$\frac{X}{Y} = \frac{1 + (2r\pi)^2}{\sqrt{(1-\pi^2)^2 + (2r\pi)^2}} = \frac{1 + (2 \times 0.52)^2}{\sqrt{(1 - \pi^2)^2 + (2 \times 0.52)^2}} = 1.3873$

$X = 0.05 \times 1.3873$   
 $X = 0.07 \text{ in.}$

$y(t) = 0.05 \sin 27.8t$

$y''(t) = \omega^2 y$

$y''(t) = \omega^2 y$

~~Force~~

~~Force~~

Spring force.

$$F_s = K(x - y)$$

$$F_d = K(0.07 \sin 2t, y - 0.05)$$

$x(t)$  is  $C \cos \omega t + D \sin \omega t$

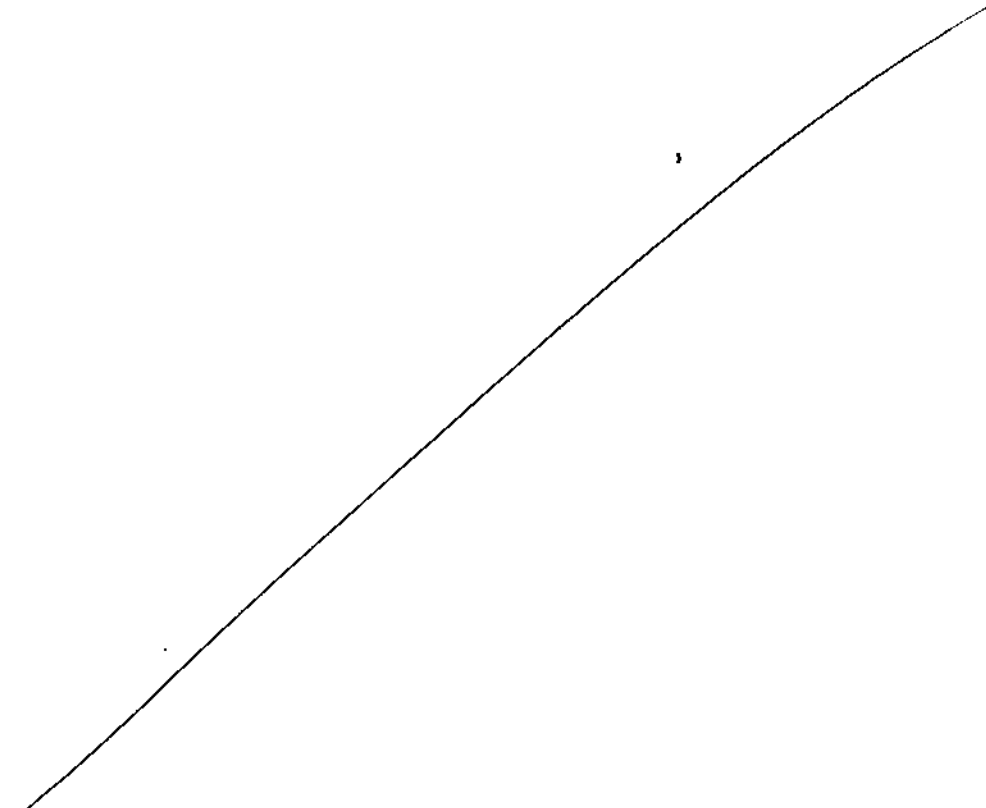
$x(t)$  is  $K \cos \omega t$ .

$$x(t) = \frac{F_0}{m} \frac{1}{\sqrt{(\omega_n^2 - \omega^2)^2 + (2\zeta \omega \omega_n)^2}} \cos(\omega t - \phi_1 - \phi_2)$$

$$F_0 = K(0.02) = 0.8$$

$$F_d = 1.5(x - y)$$

X



Damping force:

$$F_d = c(\dot{x} - \dot{y})$$

3) (15 pts) A steel beam ( $E = 30\,000\,000\text{ psi}$ ) is simply-supported, has a total length  $L = 12\text{ ft}$ , and a moment of inertia  $I = 57.6\text{ in}^4$ . It carries at its mid-span an electric motor of weight  $W = 1000\text{ lb}$  that runs at  $1800\text{ rpm}$ . Due to the unbalance, the rotor sets up a rotating centrifugal force of magnitude  $P = 500\text{ lb}$ . Neglecting the weight of the beam, calculate the amplitude of vibration in steady-state. It is known that when a unit static load is applied at mid-span, the corresponding static deflection of the beam at mid-span is  $\frac{L^3}{48EI}$ .

$E = 30\,000\,000\text{ psi}$      $L = 12\text{ ft} = 144\text{ in}$      $I = 57.6\text{ in}^4$      $W = 1000\text{ lb}$   
 $P = 500\text{ lb}$

$$K = \frac{48EI}{L^3} = \frac{48 \times 30\,000\,000 \times 57.6}{144^3} = 27777.777 \quad \checkmark$$

$$\omega_n = \sqrt{\frac{K}{m}} = \sqrt{\frac{27777.777}{1000/386.4}} = 103.6\text{ rad/s} \quad \checkmark$$

$$\omega = 1800 \frac{\text{rev}}{\text{min}} \times \frac{2\pi\text{ rad}}{1\text{ rev}} \times \frac{1\text{ min}}{60\text{ s}} = 188.5\text{ rad/s}$$

$$r = \frac{\omega}{\omega_n} = \frac{188.5}{103.6} = 1.82$$

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$$X = \frac{moe \cdot r^2}{m \sqrt{(1-r^2)^2 + (2\zeta r)^2}}$$

No damping  $\zeta = 0 \rightarrow X = \frac{moe \cdot r^2}{m \sqrt{(1-r^2)^2}} = \frac{moe \cdot r^2}{m (1-r^2)}$

$$P = moe \cdot r^2 \rightarrow 500 = moe \cdot \omega^2$$

$$moe = \frac{500}{188.5^2} = 0.01407$$

$m_{motor} = \frac{1000}{386.4} = 2.588\text{ lb/in}$

$$\Rightarrow X = \frac{0.01407 \times 1.82^2}{2.588 (1 - 1.82^2)} = 0.00778\text{ in} \quad \checkmark$$