

**MEN 330      MECHANICAL VIBRATIONS**

***HW#2 – Solution***

**Pb 2.5**

$$m = \frac{9000}{9.8}$$

Let  $\omega_n = 7.5 \text{ rad/sec}$

$$\omega_n = \sqrt{\frac{k_{eq}}{m}}$$

$$k_{eq} = m\omega_n^2 = \left[ \frac{9000}{9.8} \right] (7.5)^2 = 51658.2 \text{ N/m} = 4 \text{ k}$$

where k is the stiffness of the air spring.

$$\text{Thus } k = \frac{51658.2}{4} = 12914.5 \text{ N/m}$$

**Pb 2.13**

Let  $x_1, x_2 =$  displacement of pulleys 1 and 2

$$x = 2x_1 + 2x_2 \quad (1)$$

Let P = tension in the rope.

For equilibrium of pulley 1:  $2P = k_1 x_1 \quad (2)$

For equilibrium of pulley 2:  $2P = k_2 x_2 \quad (3)$

Where

$$\frac{1}{k_1} = \frac{1}{4k} + \frac{1}{4k} = \frac{1}{2k}, \quad k_1 = 2k$$

And  $k_2 = k + k = 2k$

Combining Eqs (1) to (3)

$$x = 2x_1 + 2x_2 = 2\left(\frac{2P}{k_1}\right) + 2\left(\frac{2P}{k_2}\right) = 4P\left(\frac{1}{2k} + \frac{1}{2k}\right) = \frac{4P}{k}$$

Let  $k_{eq} =$  equivalent spring constant of the system:

Equation of motion of mass m:  $m\ddot{x} + k_{eq}x = 0$

$$\therefore \omega_n = \sqrt{\frac{k_{eq}}{m}} = \sqrt{\frac{k}{4m}}$$

**Pb 2.21**

$$b = 2l \sin \theta$$

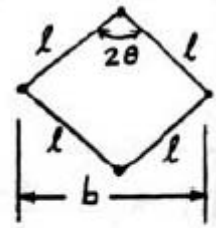
Neglect masses of links.

$$(a) \quad k_{eq} = k \left( \frac{4l^2 - b^2}{b^2} \right) = k \left( \frac{4l^2 - 4l^2 \sin^2 \theta}{4l^2 \sin^2 \theta} \right)$$

$$= k \left( \frac{\cos^2 \theta}{\sin^2 \theta} \right)$$

$$\omega_n = \sqrt{\frac{k_{eq}}{m}} = \sqrt{\frac{k g \operatorname{cosec}^2 \theta}{W}}$$

$$(b) \quad \omega_n = \sqrt{\frac{k g}{W}} \quad \text{since } k_{eq} = k.$$



(from solution of problem 1.8)

**Pb 2.33**

Assume same area of cross section for all segments of the cable. Speed of blades = 300 rpm = 5 Hz = 31.416 rad/sec.

$$\omega_n^2 = \frac{k_{eq}}{m} = (2 (31.416))^2 = (62.832)^2$$

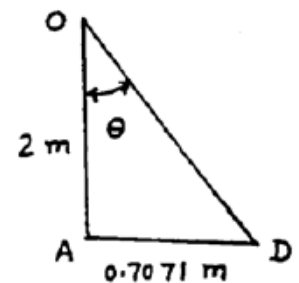
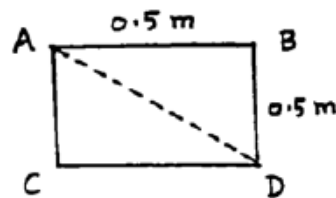
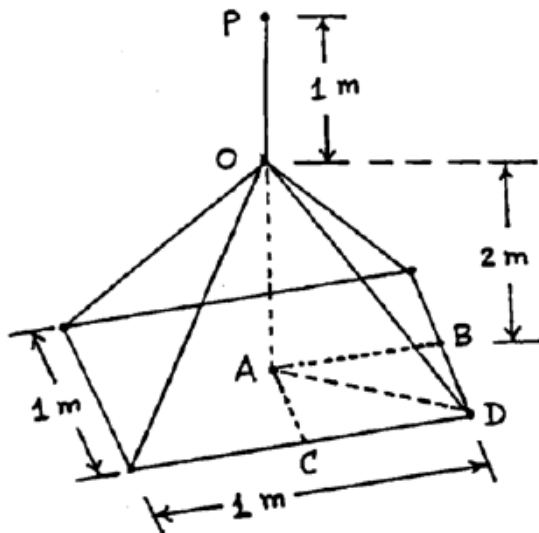
$$k_{eq} = m \omega_n^2 = 250 (62.832)^2 = 98.6965 (10^4) \text{ N/m} \quad (1)$$

$$AD = \sqrt{0.5^2 + 0.5^2} = 0.7071 \text{ m}, \quad OD = \sqrt{2^2 + 0.7071^2} = 2.1213 \text{ m}$$

Stiffness of cable segments:

$$k_{PO} = \frac{AE}{\ell_{PO}} = \frac{A (207) (10^9)}{1} = 207 (10^9) \text{ A N/m}$$

$$K_{OD} = \frac{AE}{\ell_{OD}} = \frac{A (207) (10^9)}{2.1213} = 97.5817 (10^9) \text{ A N/m}$$



$$\theta = \tan^{-1} \left( \frac{0.7071}{2} \right)$$

$$= 19.4710^\circ$$

The total stiffness of the four inclined cables ( $k_{ic}$ ) is given by:

$$k_{ic} = 4 k_{PO} \cos^2 \theta$$

$$= 4 (97.5817) (10^9) \text{ A} \cos^2 19.4710^\circ = 346.9581 (10^9) \text{ A N/m}$$

Equivalent stiffness of vertical and inclined cables is given by:

$$\frac{1}{k_{eq}} = \frac{1}{k_{PO}} + \frac{1}{k_{ic}}$$

$$\text{i.e., } k_{eq} = \frac{k_{PO} k_{ic}}{k_{PO} + k_{ic}}$$

$$= \frac{(207 (10^9) \text{ A}) (346.9581 (10^9) \text{ A})}{(207 (10^9) \text{ A}) + (346.9581 (10^9) \text{ A})} = 129.6494 (10^9) \text{ A N/m} \quad (2)$$

Equating  $k_{eq}$  given by Eqs. (1) and (2), we obtain the area of cross section of cables as:

$$A = \frac{98.6965 (10^4)}{129.6494 (10^9)} = 7.6126 (10^{-6}) \text{ m}^2$$

### Pb 2.46

Consider the springs connected to the pulleys (by rope) to be in series. Then:

$$\frac{l}{k_{eq}} = \frac{1}{k} + \frac{1}{5k}$$

$$\therefore k_{eq} = \frac{5}{6} k$$

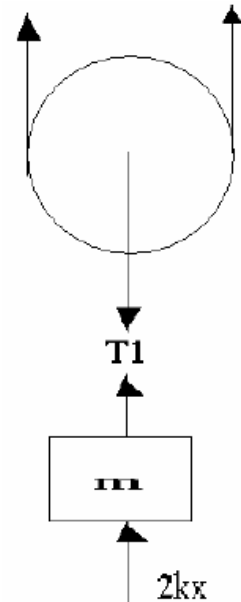
Let the displacement of mass  $m$  be  $x$ .

Then the extension of the rope (Spring Connected to the pulleys) =  $2x$ .

From the free body diagram, the equation of motion  $m$  becomes:

$$m\ddot{x} + 2kx + k_{eq} = 0$$

$$\therefore m\ddot{x} + \frac{11}{3}kx = 0$$



### Pb 2.48

$$T = \text{Kinetic energy} = \frac{1}{2} m \dot{x}^2 + \frac{1}{2} J_0 \dot{\theta}^2$$

$$U = \text{potential energy} = \frac{1}{2} k_s x^2$$

Where  $\theta = \frac{x}{r}$ ,  $x_s = \text{extension of spring} = 4r\theta = 4x$

Hence,

$$T = \frac{1}{2} \left( m + \frac{J_0}{r^2} \right) \dot{x}^2 \quad ; \quad U = \frac{1}{2} (16k)x^2$$

Using the relation  $\frac{d}{dt}(T+U) = 0$ , we obtain the equation of motion of the system as:

$$\left( m + \frac{J_0}{r^2} \right) \ddot{x} + 16kx = 0$$

### Pb 2.93

Assume that the bicycle and the boy fall as a rigid body by 5 cm at point A. Thus the mass ( $m_{eq}$ ) will be subjected to an initial downward displacement of 5 cm ( $t = 0$  assumed at point A):

$$\omega_n = \sqrt{\frac{k_{eq}}{m_{eq}}} = \sqrt{\frac{(50000)(9.81)}{800}} = 24.7614 \text{ rad/sec}$$

$x_0 = 0.05 \text{ m}, \dot{x}_0 = 0$

$$c_c = 2 m \omega_n = 2 \left[ \frac{800}{9.81} \right] (24.7614) = 4038.5566 \text{ N-s/m}$$

$$\zeta = \frac{c}{c_c} = \frac{1000.0}{4038.5566} = 0.2476 \text{ (underdamping)}$$

$$\omega_d = \omega_n \sqrt{1 - \zeta^2} = 24.7614 \sqrt{1 - 0.2476^2} = 23.9905 \text{ rad/sec}$$

Response of the system:

$$x(t) = X e^{-\zeta \omega_n t} \sin(\omega_d t + \phi)$$

$$\text{where } X = \left\{ x_0^2 + \left( \frac{\dot{x}_0 + \zeta \omega_n x_0}{\omega_d} \right)^2 \right\}^{\frac{1}{2}}$$

$$= \left\{ (0.05)^2 + \left( \frac{(0.2476)(24.7614)(0.05)}{23.9905} \right)^2 \right\}^{\frac{1}{2}} = 0.051607 \text{ m}$$

$$\text{and } \phi = \tan^{-1} \left( \frac{x_0 \omega_d}{\dot{x}_0 + \zeta \omega_n x_0} \right) = \tan^{-1} \left( \frac{0.05 (23.9905)}{0.2476 (24.7614) (0.05)} \right) = 75.6645^\circ$$

Thus the displacement of the boy (positive downward) in vertical direction is given by

$$x(t) = 0.051607 e^{-6.1309 t} \sin(23.9905 t + 75.6645^\circ) \text{ m}$$