

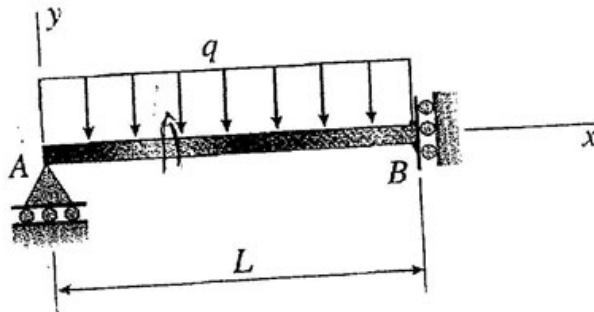
Check #1

Name: _____

MEN 302
EXAM 1

5460

- 1) (20 pts) The beam shown in the figure has a roller support at A and a guided support at B. The beam is under a uniform loading q . Use the second order differential equation to derive the equation of the deflection curve and determine the deflection at end B.



- Let's take a cut at $0 < x < L$ from A.

{ but $\sum F_y$ of the whole system at equilibrium is zero.

$$\sum F_y = 0 \Rightarrow R_A - q \times L = 0$$
$$\Rightarrow R_A = qL \quad \checkmark$$

* and $\sum M_A = 0 \Rightarrow M - R_A \times x + q \times x \times \frac{x}{2} = 0$

$$\therefore M - qLx + \frac{q x^2}{2} = 0$$

$$EI y'' = M = qLx - \frac{q x^2}{2}$$

$$\therefore EI y' = \frac{qL x^2}{2} - \frac{q}{2} \frac{x^3}{3} = \frac{qL x^2}{2} - \frac{q x^3}{6} + C_1$$

integrating y' to y

$$EI y = \frac{qL x^3}{2 \times 3} - \frac{q x^4}{6 \times 4} + C_1 x + C_2$$

for $x=0$ y (the deflection) = 0 ✓

$$0 = \frac{1}{EI} (c_2) \Rightarrow \underline{c_2 = 0}$$

for $x=0$ $y' = 0$ ✗

$$0 = \frac{1}{EI} (c_1) \Rightarrow \underline{c_1 = 0}$$

$$\therefore \boxed{y = \frac{1}{EI} \left(\frac{qLx^3}{6} - \frac{qx^4}{24} \right)}$$

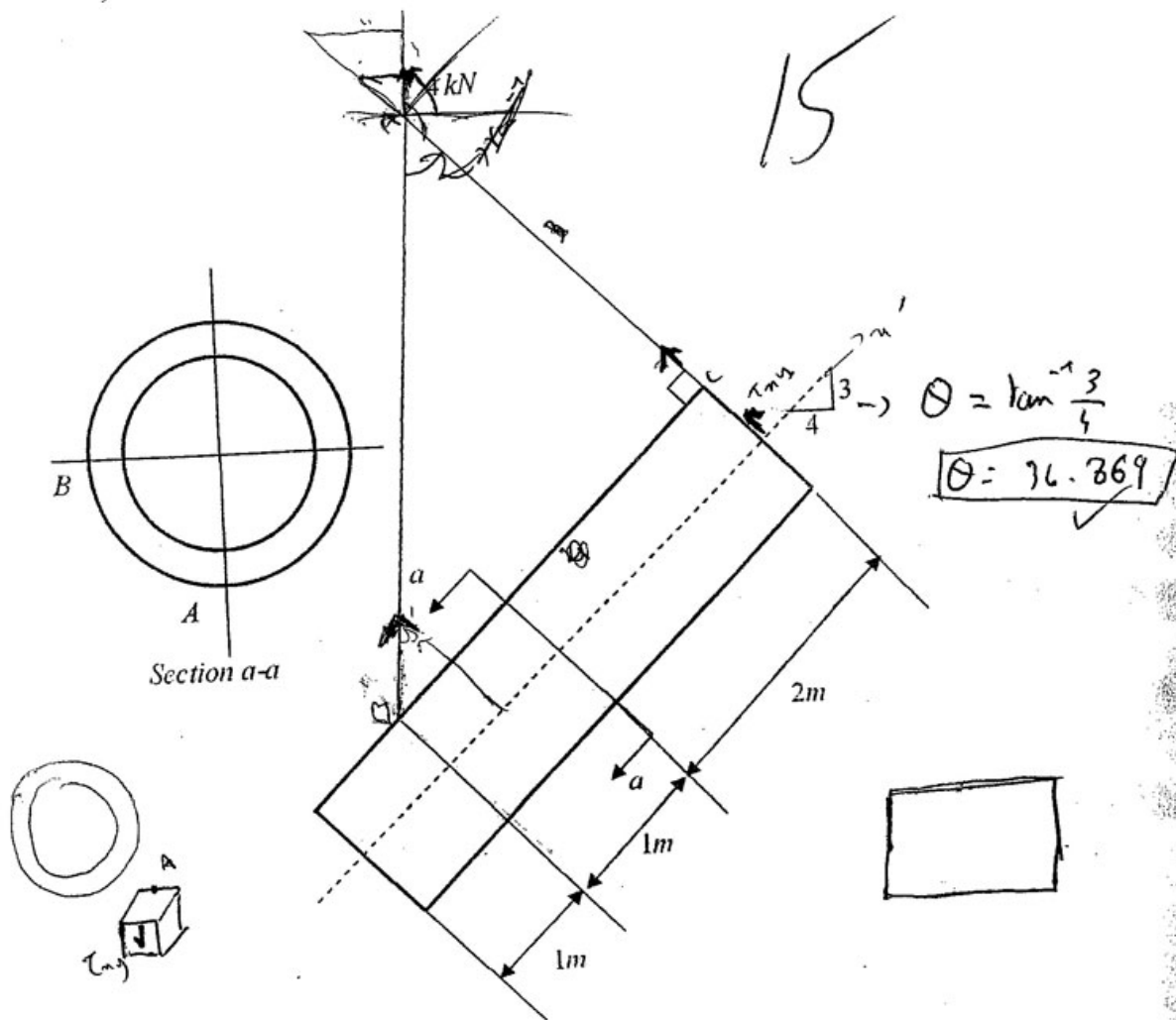
\therefore y_{avg} for $x=L$.

$$y_{\text{avg}} = \frac{1}{EI} \left(\frac{qL \cdot L^3}{6} - \frac{qL^4}{24} \right) = \frac{1}{EI} \left(\frac{qL^4}{6} - \frac{qL^4}{24} \right)$$

$$= \frac{1}{EI} \left(\frac{4qL^4 - qL^4}{24} \right) = \frac{1}{EI} \left(\frac{3qL^4}{24} \right) = \frac{1}{EI} \left(\frac{qL^4}{8} \right)$$

✗

2) (30 pts) An aircraft fuel tank, which may be considered to be a thin-walled cylindrical pressure vessel, has a mean radius $r = 300\text{ mm}$ and a thickness $t = 6\text{ mm}$. It is being hoisted by two cables into the position shown. Assuming that the vessel weighs 102 kg/m and contains fuel which exerts an internal pressure of 0.5 MPa , perform a complete stress analysis using the Von Mises criterion at points A and B on the cross-section, to calculate the factor of safety against yielding at section $a-a$ located at the middle of the tank. For each point, show the stresses on a properly oriented element. Assume that the yield strength of the material is 360 MPa . Note that for a hollow circular cross-section, the maximum bending shear stress is $\frac{2V}{A}$ where V is the shear force and A is the area of the cross-section.



4 kN along $x'y' = 4\text{ kN} \sin(90 - 53.1) = 4 \times \sin(36.869)$

$= 2.4\text{ kN} = T_{x'y'}$

No external force.

$$\delta_y' = \frac{Pr}{t} = \frac{25 \text{ MPa}}{25 \text{ MPa}}$$

$$\delta_x' = \frac{Pr}{2t} = \frac{25}{2} = 12.5 \text{ MPa}$$

$$P = 0.5 \text{ MPa} = 500000 \text{ Pa}$$

$$r = 0.3 \text{ m}$$

$$t = 0.006 \text{ m}$$

$$A = \pi$$

$$\tau_{xy}' = \frac{2.4 \text{ kN}}{A} = \frac{2.4}{282600} = 8.49 \cdot 10^{-6}$$

δ_x'

$$12.5 = \frac{1}{2} (\delta_x + \delta_y) + \frac{1}{2} (\delta_x - \delta_y) \cos 2\theta + \tau_{xy} \sin 2\theta$$

$$12.5 = \frac{1}{2} (\delta_x + \delta_y) + \frac{1}{2} (\delta_x - \delta_y) \cos(73.74) + \tau_{xy} \sin(73.74)$$

$$12.5 = 0.5 \delta_x + 0.5 \delta_y + (0.5 \times 0.28) \delta_x - (0.5 \times 0.28) \delta_y + \tau_{xy} \times 0.96$$

$$12.5 = \delta_x (0.5 + 0.14) + \delta_y (0.5 - 0.14) + 0.96 \tau_{xy}$$

$$12.5 = 0.64 \delta_x + 0.36 \delta_y + 0.96 \tau_{xy} \quad \text{①} \quad \text{Why this?}$$

$$\delta_y' = \frac{1}{2} (\delta_x + \delta_y) - \frac{1}{2} (\delta_x - \delta_y) \cos 2\theta - \tau_{xy} \sin 2\theta$$

$$25 = 0.5 \delta_x + 0.5 \delta_y - [0.5 \times 0.28 \delta_x - 0.5 \times 0.28 \delta_y] - 0.96 \tau_{xy}$$

$$25 = \delta_x (0.5 - 0.14) + \delta_y (0.5 + 0.14) - 0.96 \tau_{xy} \quad \text{②}$$

$$\tau_{x'y'} = -\frac{1}{2} (\delta_x - \delta_y) \sin 2\theta + \tau_{xy} \cos 2\theta$$

$$2.4 = -0.5 \times 0.96 \delta_x + 0.5 \times 0.96 \delta_y + 0.28 \tau_{xy}$$

$$\frac{2.4}{A} = -0.48 \delta_x + 0.48 \delta_y + 0.28 \tau_{xy} \quad \text{③}$$

3 equations and 3 unknowns. we calculate and found?

$$\delta_x = +17.69 \text{ MPa}$$

$$\delta_y = +20.90 \text{ MPa}$$

$$\tau_{xy} = -0.28 \text{ MPa}$$

using the principal stresses equations

$$\sigma_{1,2} = \frac{\delta_x + \delta_y}{2} \pm \sqrt{\left(\frac{\delta_x - \delta_y}{2}\right)^2 + \tau_{xy}^2}$$